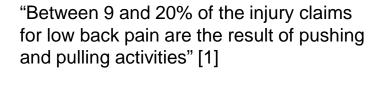


Flexible Luggage

Samer Abdelmoty, Bill Chang, Martin Phelan, Xuerui Gu

Problem

932 Million passengers flew domestically and internationally in 2016 [2]







Problem

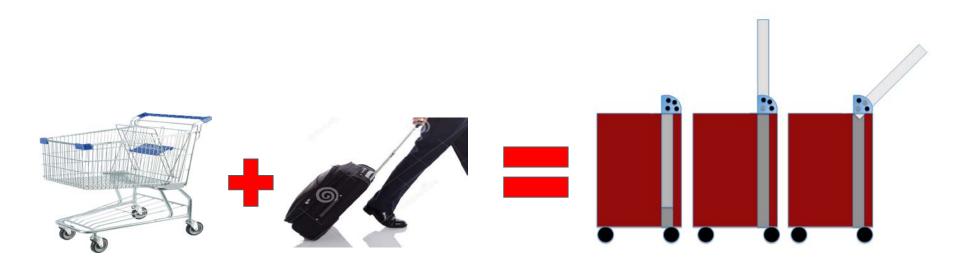








Solution



(Critical Stresses)

- Given the typical use cases of luggage bags, we perceive the most important stresses to analyze in the system include:
 - Static stress under horizontal and vertical loading conditions
 - Fatigue stress
- The overall life of a suitcase depends on how frequently it is used (i.e. how frequently the user travels)
 - For someone who travels infrequently: 10 years or more
 - For someone who travels frequently: Roughly 5 years (typical warranty period)



Materials and Manufacturing Processes

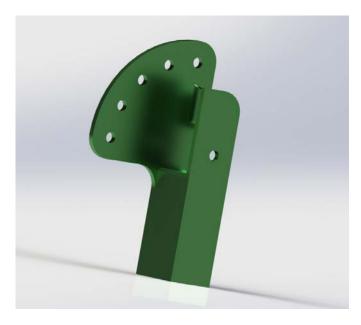


- Handle Rods and Sleeves:
 - o Aluminum, Extrusion
 - Under some load; Uniform Cross-Section and Thin-Walled



- Rod End Caps
 - ABS Plastic, Injection Molded
 - Not under significant load, but complex geometry
- End Cap Pins
 - Aluminum, Die Cast
 - Under high load, has complex geometry

Materials and Manufacturing Processes



P-Bracket

- Aluminum, Permanent Mold Casting
- Under high load; Complex geometry with undercuts
- With some redesigning could possibly be die-casted

Pivot Holder

- o ABS Plastic, Injection Molded
- Not under significant load, but complex geometry with undercuts



Hand calculation

Strength criteria:

worst case load: the user apply weight to the luggage handle

$$F = G = 1130 N$$

$$A = td = t(0.005 m)$$

$$\sigma = \frac{F}{A} = \frac{1130}{0.005t} \le \frac{\sigma_y}{FoS} = \frac{280 MPa}{2}$$

$$t \ge 0.8 mm$$

Fatigue criteria:

For Aluminum 6061-T6, for number of cycles 1.0e+06, the endurance stress under zero mean stress is 55.5 MPa. worst case loading: the user press and lift the luggage handle reversely with his weight.

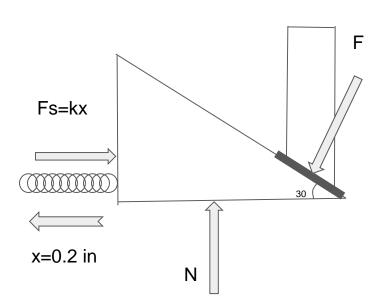
$$F = G = 1130 N$$

$$A = td = t(0.005 m)$$

$$\sigma_y = \frac{F}{A} = \frac{1130}{0.005t} \le \frac{\sigma_e}{FoS} = \frac{55.5 MPa}{2}$$

$$t \ge 4.0 mm$$

Product: Spring Design



$$F_1 = \cos(30) * 5 lbs = 4.33 lbs$$

$$F_{s1} = \frac{F_1}{2} = 2.17 \ lbs$$

$$k_1 = \frac{F_{s1}}{x_1} = \frac{8.66 \ lbs}{0.2 \ in} = 10.8 \ lb/in$$

$$F_2 = \cos(30) * 20 \ lbs = 17.32 \ lbs$$

$$F_{s2} = \frac{F_2}{2} = 17.32 \ lbs$$

$$k_2 = \frac{F_{s2}}{x_2} = \frac{17.32 \ lbs}{0.2 \ in} = 43.3 \ lb/in$$

Specific Design Constraints

- Functionality
 - Must retain height adjustment mechanism
 - Must withstand loading conditions that luggage bags typically go through

Geometry

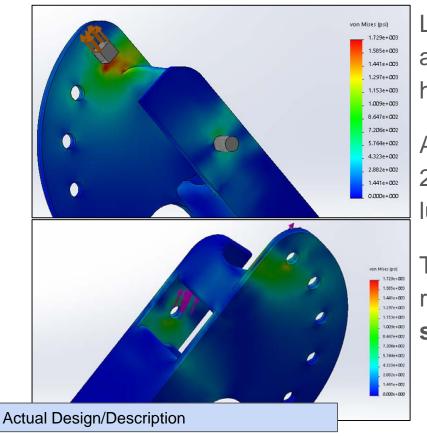
 For prototyping purposes, the dimensions of custom components had to be compatible with the other suitcase components

Aesthetics

Minimal changes to outward appearance of suitcase



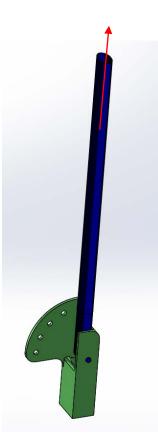
Stress Analysis of P-Bracket



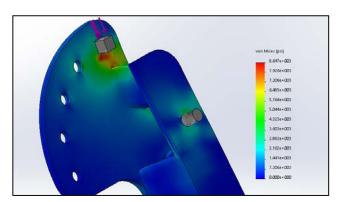
Loading scenario: the user attempts to lift the bag using the handle

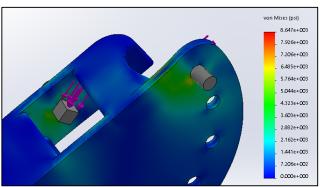
Assume the bag has a mass of 23 kg (The maximum allowed luggage weight on airlines)

The peak stress in the bracket is roughly 1.7 kpsi, which yields a safety factor of about 5.2



Stress Analysis of P-Bracket

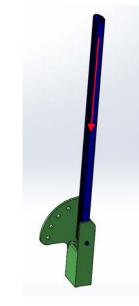




Loading scenario: the user puts all his weight (250 lbs) on the handle

The peak stress in the bracket is roughly 8.6 kpsi, which yields a **safety factor of about 1**

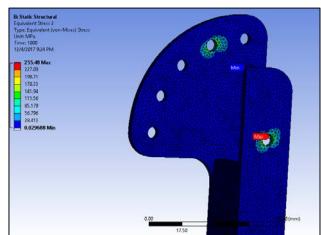
However, we believe that this loading scenario is somewhat unrealistic

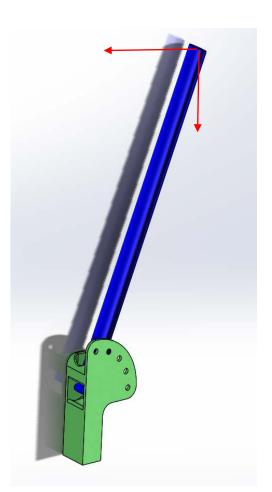




Position 1:Stress Analysis

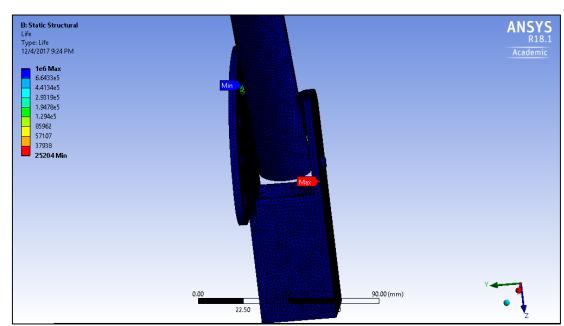
- 115 N horizontal force applied to the handle
- 50N Vertical force applied to the handle
- Housing material: Al Alloy
- Max Principal Stress: 255.4 MPa Safety Factor: 1.10





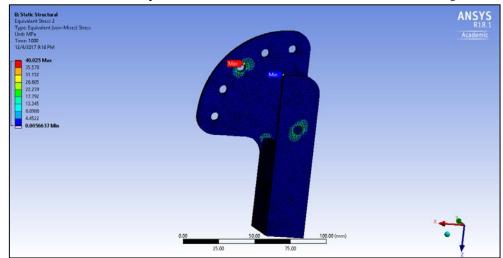
Position 1: Fatigue Analysis

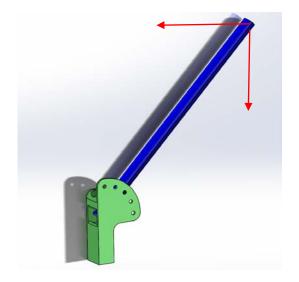
- 115 N horizontal force applied to the handle
- 50N Vertical force applied to the handle
- Zero based loading
- Housing material: Al Alloy
- Life: 25204 cycles



Position 2: Stress Analysis

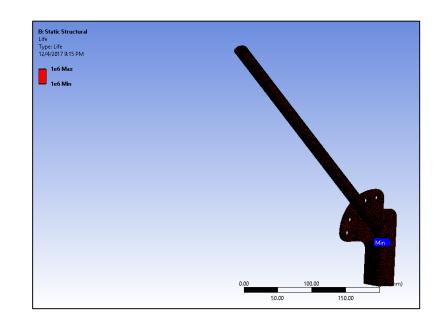
- 115 N horizontal force applied to the handle
- 50N Vertical force applied to the handle
- Housing material: Al Alloy
- Max Principal Stress: 40.02 MPa Safety Factor: 7





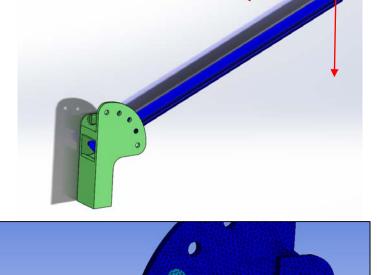
Position 2: Fatigue Analysis

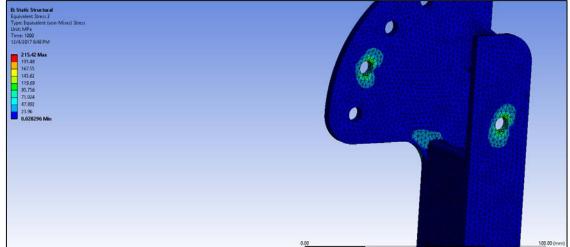
- 115 N horizontal force applied to the handle
- 50N Vertical force applied to the handle
- Zero based loading
- Housing material: Al Alloy
- Life: 1e6 cycles



Position 3: Stress Analysis

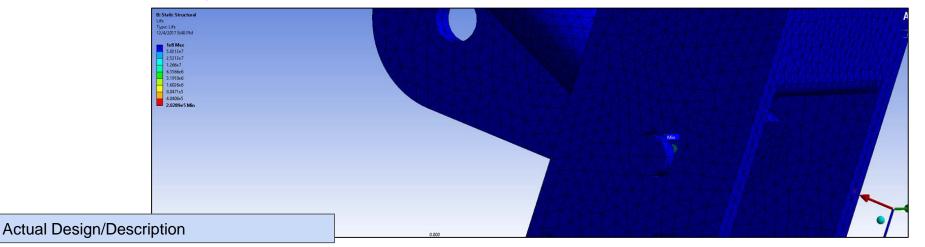
- 115 N horizontal force applied to the handle
- 50N Vertical force applied to the handle
- Housing material: Al Alloy
- Max Principal Stress: 215.152MPa
- Safety Factor: 1.3





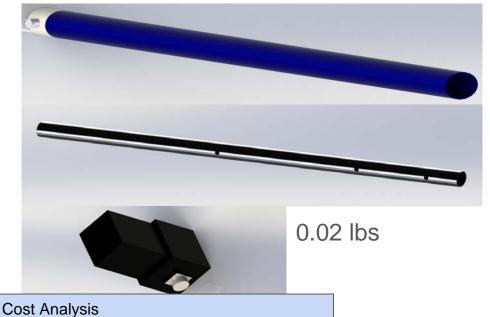
Position 3: Fatigue Analysis

- 115 N horizontal force applied to the handle
- 50N Vertical force applied to the handle
- Zero based loading
- Housing material: Al Alloy
- Life: 2e5 cycles



Cost Analysis: Handle Rods and Sleeves

- Aluminum Extrusion
- Tooling Cost: ~\$1200
- Material Cost: ~0.94 USD/lb



0.15 lbs

0.18 lbs

Total Cost



| (per unit) | handles = \$0.66/unit |
|---------------------|---------------------------------|
| Total Material Cost | \$0.94/lb * 0.35 lbs * 2 |
| Total Tooling Cost | \$1200 * 3 profiles = \$3600 |

\$3600 + 0.66 * n

Cost Analysis: Rod End Pieces and Pivot Holder

- Plastic Injection Molding (ABS)
- Tooling Cost: ~\$12,000
- Material Cost: ~\$1.20/lb



0.02 lbs





Cost Analysis

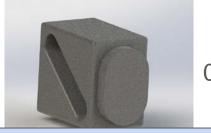
| Total Tooling Cost | \$12,000 * 3 patterns = \$36,000 |
|--------------------------------|--|
| Total Material Cost (per unit) | \$1.20/lb * 0.04 lbs * 2 handles = \$0.096/unit |
| Total Cost | \$36,000 + 0.096 * n |

Cost Analysis: Rod End Piece Pins

- Aluminum Die-Casting
- Tooling Cost: ~\$4,000
- Material Cost: ~\$0.94/lb



0.01 lbs



0.06 lbs

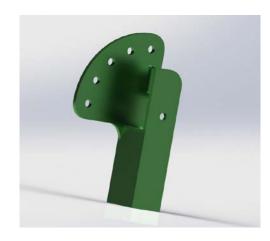


| (10.11.11) | = \$0.066/unit |
|--------------------------------------|----------------|
| | 0.07 lbs * 2 |
| Total Tooling Cost \$4,000 * \$8,000 | 2 patterns = |

Cost Analysis

Cost Analysis: P-Bracket

- Aluminum Investment-Casting
- Wax Mold:
 - Assume same fixed cost as if injection molding
 - Known Quantities:
 - Projected Area: 7 in * 9.5 in = 66.5 in2
 - Part Depth = 1.47 in
 - SP = 6
 - Fixed Cost: \$22,593.73



Cost Analysis: P-Bracket (cont.)

- Aluminum Investment-Casting
- Variable Cost:
 - Assume same variable cost as if sand casting
 - Known quantities:
 - N = n
 - V = 3.57 in
 - A = 7 in * 9.5 in = 66.5 in 2
 - Density = 0.098 lbf/in3
 - Price of material = \$0.94/lb
 - Assume 2% waste
 - \circ Material Cost = (1.02)(n)(0.94)(0.098)(3.57) = 0.335n
 - Setup = n
 - \circ Ladling = (0.03)*n = 0.03n
 - \circ Cooling = (0.008)(n)(66.5) = 0.532n
 - Variable Cost = 1.897n



| | Total Cost | \$22,593.73 + 3.794 * n |
|--|--------------------------------|-------------------------------------|
| | Total Material Cost (per unit) | \$1.897* 2 handles = \$3.794unit |
| | Total Tooling Cost | \$22,593.73 |

Cost Analysis Summary

Assumptions for fixed cost:

| Total Tooling Cost | \$22,593.73 | Total Tooling Cost | \$12,000 * 3 patterns = \$36,000 |
|--------------------------------|---|--------------------------------|--|
| Total Material Cost (per unit) | \$1.897* 2 handles = \$3.794/unit | Total Material Cost (per unit) | \$1.20/lb * 0.04 lbs * 2 handles = \$0.096/unit |
| Total Cost | \$22,593.73 + 3.794 * n | Total Cost | \$36,000 + 0.096 * n |
| Total Tooling Cost | \$1200 * 3 profiles = \$3600 | Total Tooling Cost | \$4,000 * 2 patterns = \$8,000 |
| Total Material Cost (per unit) | \$0.94/lb * 0.35 lbs * 2 handles = \$0.66/unit | Total Material Cost (per unit) | \$0.94/lb * 0.07 lbs * 2 handles = \$0.066/unit |
| Total Cost | \$3600 + 0.66 * n | Total Cost | \$8,000 + 0.066 * n |

Cost Analysis

\$70,193.73 + 4.616n

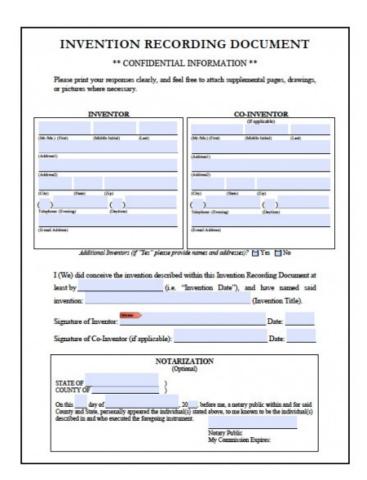
Conclusion

- For a production run of 100,000 units, we have a unit cost of \$5.32 per luggage bag
 - Only includes cost of handle mechanism
 - The price of a luggage bag varies between \$50-\$300
 - The "Flexible Luggage" handle system increases the cost of luggage bags by
 \$7.50 assuming a 40% profit margin



Next Steps

- File Invention Disclosure
- Planning to build a more robust design
 - Better aesthetics
 - Modular design; can be fitted onto different models of luggage bags



Questions?

Appendix

